

STATIC ANALYSIS OF ASYMMETRIC MULTISTORY STRUCTURES

by

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SYNOPSIS

A simplified criterion for the computation of dynamic amplification factors for twisting moments in multistory systems is proposed. By means of a generalized-coordinates approach and proper approximations to the first translational, torsional and combined modes, applicability of results valid for single-story systems is extended to multi-story structures. The accuracy of the approximate procedure is calibrated by comparing its results with those produced by standard modal analysis.

INTRODUCTION

Seismic static analysis of multistory structures is usually performed under the assumptions that their mass is lumped at floor levels and that the effects of the design earthquake are represented by a set of lateral forces, each acting at center of gravity of one of the masses. Implicit in the criterion is the assumption that torsional oscillations are accounted for by the static torsional eccentricities at each story (that is, the lever arm of the static story shear with respect to the twist center) multiplied by suitable amplification factors. The criterion breaks down, for instance, in buildings where nominal static eccentricities differ drastically among stories, as in that case torsional oscillations induced by the largest static eccentricities give place to inertia forces that may produce significant dynamic eccentricities throughout the structure. This paper presents and calibrates a method for determining a set of dynamic torsional couples, assumed to act the floor masses, and such that story torsional moments computed by static equilibrium of the portion of building above a given story, under the action of floor lateral forces and torsional couples, provide an approximate value of the actual dynamic story torsion. The criterion may be applied when story shears and eccentricities are obtained either from a static analysis or from a dynamic analysis that neglects torsional degrees of freedom.

STATIC TORSION

A definition of static torsion that differs from that ordinarily used is introduced as follows. Let Q_i be the lateral force acting at the center of gravity of mass M_i , and J_i the moment of inertia of M_i with respect to a vertical axis going through its center of gravity. Let θ_i and δ_i be respectively the rotation of mass M_i in a horizontal plane and the translation of its center of gravity in the direction parallel to Q_i . *By definition*, the static torsional couple T_i , assumed acting on mass M_i , is related to Q_i by the following proportionality:

$$\frac{T_i}{Q_i} = \frac{J_i \theta_i}{M_i \delta_i} \quad (1)$$

DYNAMIC TORSION

An approximate estimate of dynamic torsion can be obtained if T_i as given by eq 1, is multiplied by an adequate amplification factor. A criterion for defining such a factor is derived in the following, by extension of the results presented by Newmark and Rosenblueth (1971) for simple story systems.

Let us start by defining a reference system, as shown either in fig 1a or 1b. It is an asymmetric single story structure with mass M , assumed concentrated at the top, center of gravity g , and moment of inertia J with respect to a vertical going through g . Let the story shear stiffness in the assumed direction of seismic forces be denoted by K_{11} , the twist center by t and the story torsional stiffness with respect to g by K_{22} ; e is the nominal eccentricity. The static torsion is defined according to eq 1. From K_{11} , K_{22} and e it is easy to obtain the off-diagonal stiffness K_{12} .

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Let Z_i , $i=1,2$, be the characteristic vectors of the system and z_1, z_2 the components of either of them. The conditions for free vibration can be expressed as follows:

$$-\omega^2 \begin{bmatrix} M & \\ & J \end{bmatrix} \begin{Bmatrix} z_1 \\ z_2 \end{Bmatrix} + \begin{bmatrix} K_{11} & K_{12} \\ K_{21} & K_{22} \end{bmatrix} \begin{Bmatrix} z_1 \\ z_2 \end{Bmatrix} = 0 \quad (2)$$

after premultiplying by $Z^T = [z_1 \ z_2]$, one obtains, after making $z_1 = 1, z_2 = \alpha$,

$$-\omega^2 (M + J\alpha^2) + K_{11} + (K_{12} + K_{21})\alpha + K_{22}\alpha^2 = 0 \quad (3)$$

In order to define a correspondence of the generalized representation of a multistory system and a reference system, denote by Z_1 and Z_2 the first two characteristic vectors, and by Z_δ and Z_θ respectively the shapes (or approximations thereof) of the fundamental modes in pure shear and pure torsion (assuming all centers of gravity linearly fixed) of the multistory system under study. If the approximation $Z^T = [Z_\delta^T \ \alpha Z_\theta^T]$ is introduced, the free vibration conditions can be written as

$$-\omega^2 \begin{bmatrix} \tilde{M} & \\ & \tilde{J} \end{bmatrix} \begin{Bmatrix} Z_\delta \\ \alpha Z_\theta \end{Bmatrix} + \begin{bmatrix} \tilde{K}_{11} & \tilde{K}_{12} \\ \tilde{K}_{21} & \tilde{K}_{22} \end{bmatrix} \begin{Bmatrix} Z_\delta \\ \alpha Z_\theta \end{Bmatrix} = 0 \quad (4)$$

where $\tilde{M}, \tilde{J}, \tilde{K}_{11}, \tilde{K}_{12}, \tilde{K}_{21}$ and \tilde{K}_{22} are matrices.

Premultiplying by Z^T

$$-\omega^2 (Z_\delta^T \tilde{M} Z_\delta + \alpha^2 Z_\theta^T \tilde{J} Z_\theta) + Z_\delta^T \tilde{K}_{11} Z_\delta + (Z_\delta^T \tilde{K}_{21} Z_\delta + Z_\theta^T \tilde{K}_{12} Z_\theta)\alpha + Z_\theta^T \tilde{K}_{22} Z_\theta \alpha^2 = 0 \quad (5)$$

The correspondence with the reference system is obvious. Hence, amplification results available for the single-story system can be extended to the multistory system.

Because a simplified procedure is aimed for, where no conventional computation of modes Z_δ and Z_θ is required, a reasonable shape must be assumed for them: perhaps linear, or proportional to the static displacements under the Q_i 's. Determination of the generalized masses $Z_\delta^T \tilde{M} Z_\delta$ and $Z_\theta^T \tilde{J} Z_\theta$ is straightforward and does not make use of quantities not obtained in ordinary practice. Computation of generalized stiffnesses is expressed in terms of stiffness submatrices, but can easily be expressed in terms of virtual works: take for instance $Z_\delta^T \tilde{K}_{21} Z_\delta = Z_\theta^T Q_\delta$, where Q_δ is the vector of forces associated with the approximation to the translational mode. The corresponding virtual work can be determined in terms of either internal or external forces.

The method proposed is as follows:

- a) Lateral forces Q_i are applied; δ_i, θ_i and T_i are computed.
- b) Generalized mass and stiffness matrices, as used in eq 5, are obtained.
- c) Dynamic amplification factor of static torsional couple T_E with respect to the center gravity of the floor mass in a single story structure and the corresponding ratio of dynamic to static shears are determined, either by means of approximate expressions or through application of figures such as 2-6, to be discussed.
- d) Factors determined as in the previous paragraph are applied to all T_i 's and Q_i 's, respectively.

DYNAMIC AMPLIFICATION OF TORSION FOR DESIGN PURPOSES

Attention is focused again on fig 1. The dynamic amplification factor of torsion is traditionally computed as the ratio of the dynamic eccentricity (dynamic torsional moment T_D^* with respect to the twist center divided by dynamic shear V_D) to the static eccentricity e . For this purpose, T_D^* and V_D are obtained by superposition of the corresponding modal contributions according to the following equation (Newmark and Rosenblueth, 1971):

$$R^2 = \sum_i R_i^2 + \sum_{i \neq j} \frac{R_i R_j}{1 + \epsilon_{ij}^2} \quad (6)$$

Here, R is the response of interest, R_i the corresponding contribution of the i -th mode (with a properly chosen sign) and ϵ_{ij} a quantity that accounts for the stochastic correlation between the rimes at which R_i and R_j occur. Figures 2 and 3 were obtained in accordance with the above criterion, both for a hyperbolic and for constant acceleration spectrum, taking R equal successively to the dynamic lateral force Q_D and the dynamic torsional couple T_D^* with respect to the twist center. However, in order to compare these figures with figs 4-6, they are plotted in terms of Q_D and T_D , referred to the center of mass. It is implied, for instance, that design shear forces V_1 and V_2 for the respective diaphragms in fig 1a should be obtained from considerations of static equilibrium in terms of Q_D and T_D . Results clearly differ from the case when R in eq 6 is either V_1 or V_2 , that is, when modal superposition is directly performed for internal forces at the diaphragms, because "maximum probable" values of Q_D and T_D^* or T_D do not occur simultaneously and because, in addition, V_1 and V_2 do not necessarily take place simultaneously with either of them. Figs 4 and 5 show what is obtained when V_1 and V_2 are directly computed with eq 6 and Q_D and T_D are obtained from them by statics.

Discrepancies are obvious. Take for instance the case when $\eta = 1$ and $e^2 M/J = 0.03$. From figs 2 and 3 obtains $V_1 = 0.28$, $V_2 = 0.61$, whereas from figs 4 and 5, $V_1 = 0.5$ and $V_2 = 0.5$. Suppose that stiffnesses were given by the elements shown in fig 1b, instead of those in fig 1a, and that Q_D and T_D are obtained by statics from the values of the shears acting on each diaphragm determined directly by means of eq 6. The ratio of dynamic to static shear is in this case equal to that given by fig 2. The ratio of dynamic to static torsional couples, however, differs from fig 3; it is shown in fig 6. Discrepancies observed call the attention to the need for revising usual criteria for the specification of design torsional eccentricities. This, however, does not affect the validity of the comparison made in the sequel between "exact" and approximate values of the response of asymmetric multistory structures.

RESPONSE OF MULTISTORY SYSTEMS

The approximate procedure described above was calibrated by comparing its results with the "exact" values obtained through application of modal analysis (with one translational and one torsional degrees of freedom per floor) to a number of multistory systems, covering a reasonably wide range of mechanical properties. Only the constant acceleration spectrum was considered. All stories in all systems were supposed to have a plan similar to fig 1a, with story stiffness and eccentricity (defined as the distance from the center of figure to the center of stiffness) as given in table 1. All masses were equal in a given system and b/a was taken as 2 in all cases.

"Exact" values of dynamic lateral forces and torsional couples acting at the centers of mass of all floors were obtained by statics from the "exact" design shears at both walls at all stories, directly computed by means of eq 6. Ratios of "exact" to approximate values are shown in fig 7. It is seen that the proposed method leads to design shears which are in general greater than those given by exact modal analysis, but deviations are clearly within tolerable limits for practical applications.

CONCLUSIONS

Dynamic torsional response of multistory systems can be predicted by extrapolation of results applicable to single-story systems. Definitions of generalized masses, moments of

inertia, stiffnesses and static torsions that permit such extrapolation are proposed. Story shears and twisting moments for design must be defined in such a manner as to lead, through conventional methods of structural analysis, to adequate design values of forces acting on the resisting elements.

REFERENCE

1. Newmark, N M and Rosenblueth, E, *Fundamentals of earthquake engineering*, Prentice Hall, Englewood Cliffs, N. J. (1971)

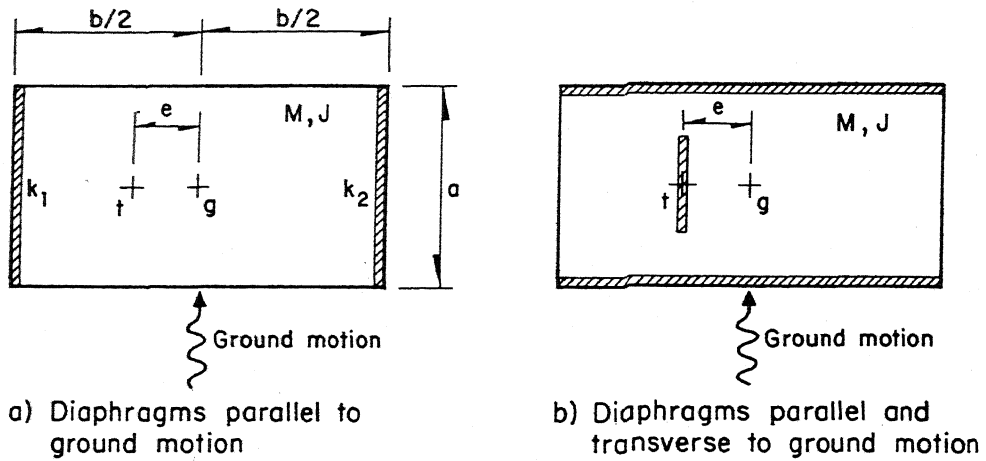


Fig 1. Reference single-story systems

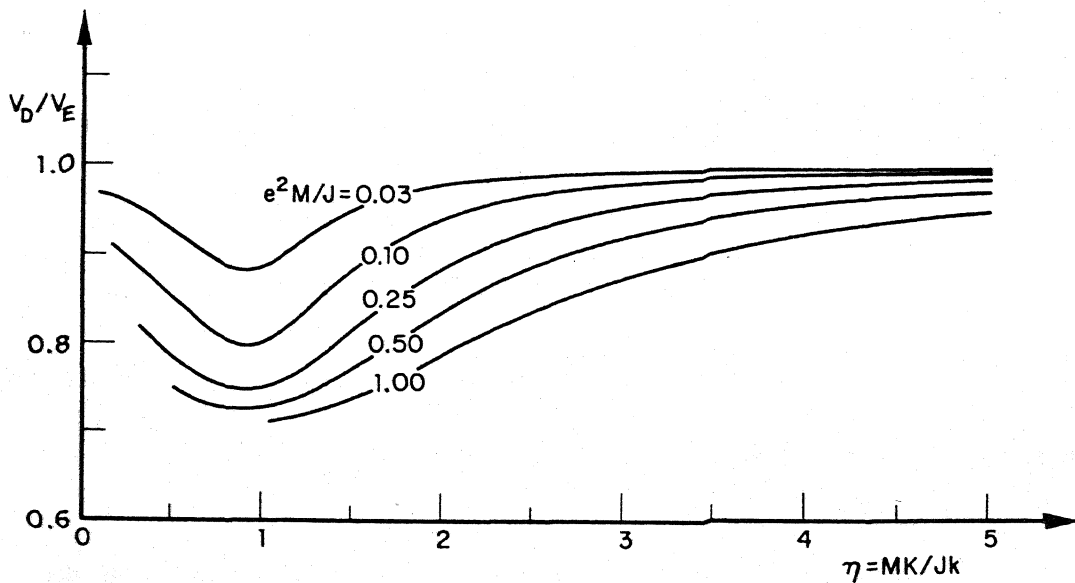


Fig 2. Ratio of dynamic to static shear. System 1b. Constant acceleration spectrum

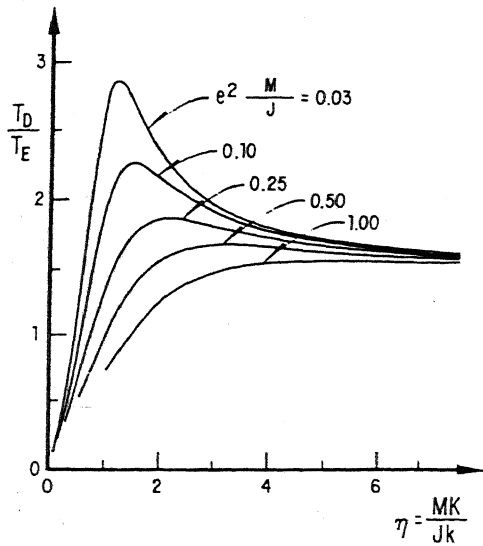


Fig 3. Ratio of dynamic to static torsion. System 1b. Constant acceleration spectrum

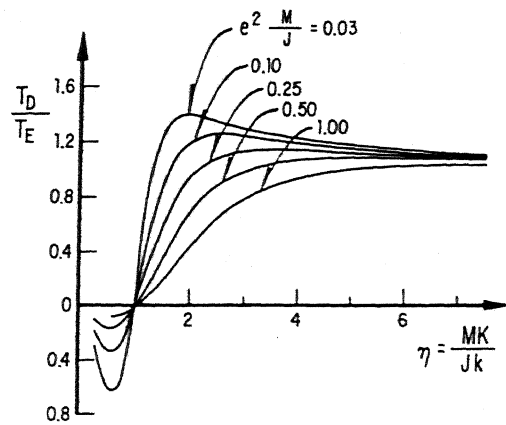


Fig 4. Ratio of dynamic to static torsion according to proposed mode-superposition criterion System 1a. Constant acceleration spectrum

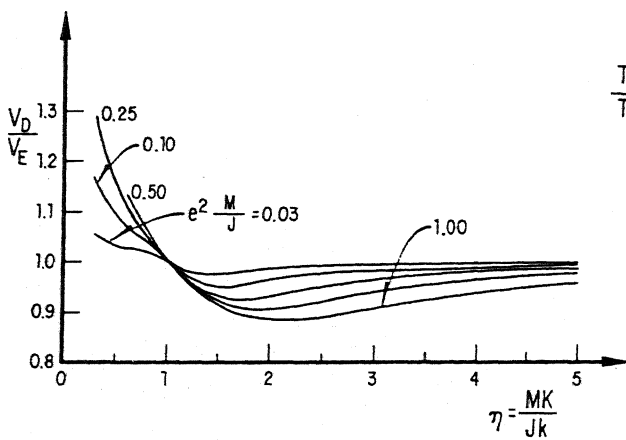


Fig 5. Ratio of dynamic to static shear according to proposed mode-superposition criterion. System 1a Constant acceleration spectrum

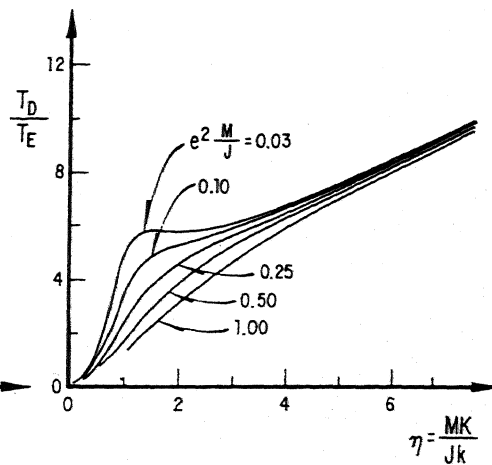
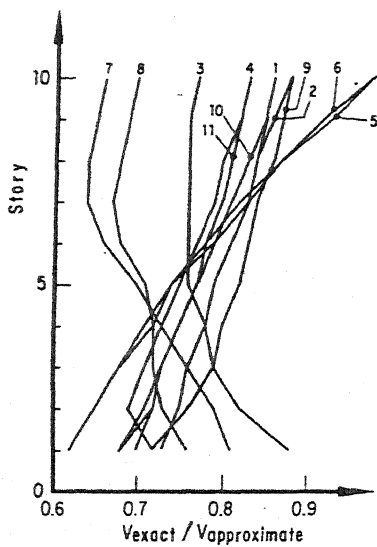
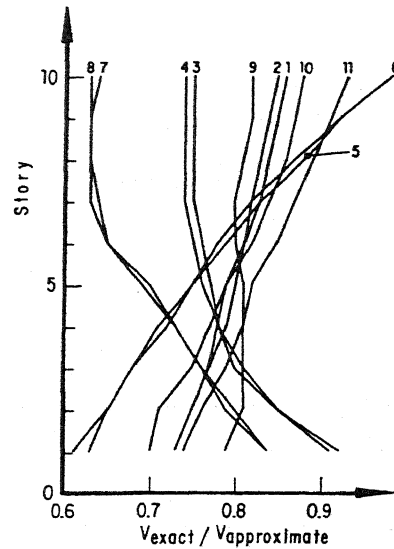


Fig 6 Ratio of dynamic to static torsion according to proposed mode-superposition criterion. System 1b. Constant acceleration spectrum



a) Stiffer diaphragm



b) Softer diaphragm

Fig 7. Ratio of exact to approximate diaphragm shear

TABLE 1. SYSTEMS STUDIED

system story	1		2		3		4		5		6		7		8		9		10		11	
	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b	K	e/b
1	1.0	0.1	1.0	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.1	1.0	0.3	1.0	0.0	1.0	0.3	1.0	0.3
2	1.0	0.1	1.0	0.3	0.9	0.1	0.9	0.3	1.0	0.1	1.0	0.3	1.0	0.1	1.0	0.3	1.0	0.0	1.0	0.3	1.0	0.0
3	1.0	0.1	1.0	0.3	0.8	0.1	0.8	0.3	1.0	0.1	1.0	0.3	1.0	0.1	1.0	0.3	1.0	0.0	1.0	0.3	1.0	0.0
4	1.0	0.1	1.0	0.3	0.7	0.1	0.7	0.3	1.0	0.1	1.0	0.3	1.0	0.1	1.0	0.3	1.0	0.0	1.0	0.3	1.0	0.0
5	1.0	0.1	1.0	0.3	0.6	0.1	0.6	0.3	1.0	0.1	1.0	0.3	1.0	0.1	1.0	0.3	1.0	0.0	1.0	0.3	1.0	0.0
6	1.0	0.1	1.0	0.3	0.5	0.1	0.5	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.3	1.0	0.0	1.0	0.0
7	1.0	0.1	1.0	0.3	0.4	0.1	0.4	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.3	1.0	0.0	1.0	0.0
8	1.0	0.1	1.0	0.3	0.3	0.1	0.3	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.3	1.0	0.0	1.0	0.0
9	1.0	0.1	1.0	0.3	0.2	0.1	0.2	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.3	1.0	0.0	1.0	0.0
10	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.1	1.0	0.3	0.1	0.1	0.1	0.3	1.0	0.3	1.0	0.0	1.0	0.0